

PID REGULATOR

6th homework

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1 Assignment [1]

A closed-loop control system is described by the following block diagram:

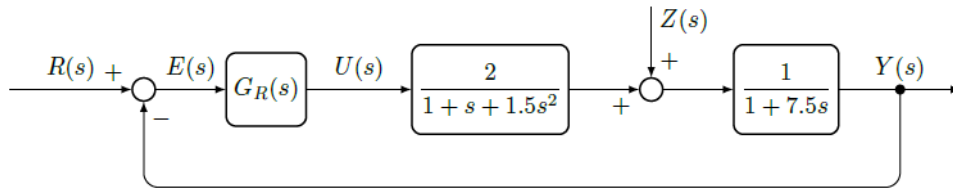


Figure 1: Closed-loop system

Regulators used in the assignment and their corresponding transfer functions are summarized below:

- PI: $G_R(s) = K_R(1 + \frac{1}{T_I s})$,
- PID (ideal): $G_R(s) = K_R(1 + \frac{1}{T_I s} + T_D s)$,
- PID (real): $G_R(s) = K_R(1 + \frac{1}{T_I s} + \frac{T_D s}{1 + T_D s})$.

1.1 Parametrizing a PI regulator with respect to $r(t)$

We assume the following substitutions:

- $G_R(s) = K_R(1 + \frac{1}{T_I s})$,
- $G_1(s) = \frac{2}{1.5s^2 + s + 1}$,
- $G_2(s) = \frac{1}{7.5s + 1}$.

The error signal with respect to a step reference signal $r(t)$ needs to be calculated:

$$E(s) = R(s) - Y(s), \quad (1)$$

$$E(s) = R(s) - \frac{G_R(s)G_1(s)G_2(s)}{1 + G_R(s)G_1(s)G_2(s)}R(s), \quad (2)$$

$$E(s) = \frac{1}{1 + G_R(s)G_1(s)G_2(s)}R(s), \quad (3)$$

$$E(s) = \frac{1}{1 + K_R(1 + \frac{1}{T_I s})\frac{2}{1.5s^2+s+1}\frac{1}{7.5s+1}}\frac{1}{s}, \quad (4)$$

$$E(s) = \frac{1}{s + K_R(s + \frac{1}{T_I})\frac{2}{1.5s^2+s+1}\frac{1}{7.5s+1}}, \quad (5)$$

$$E(s) = \frac{1}{s + \frac{2K_R(s + \frac{1}{T_I})}{11.25s^3+9s^2+8.5s+1}}, \quad (6)$$

$$E(s) = \frac{11.25s^3 + 9s^2 + 8.5s + 1}{11.25s^4 + 9s^3 + 8.5s^2 + (1 + 2K_R)s + 2\frac{K_R}{T_I}}, \quad (7)$$

$$E(s) = \frac{11.25T_I s^3 + 9T_I s^2 + 8.5T_I s + T_I}{11.25T_I s^4 + 9T_I s^3 + 8.5T_I s^2 + (1 + 2K_R)T_I s + 2K_R}, \quad (8)$$

Obtained $E(s)$ has a form $E(s) = \frac{c_0 + c_1 s + \dots + c_{n-1} s^{n-1}}{d_0 + d_1 s + \dots + d_n s^n}$ and is of the 4th order. Therefore, the ISE criterion has the form:

$$I_{3,4} = \frac{c_3^2(-d_0^2 d_3 + d_0 d_1 d_2) + (c_2^2 - 2c_1 c_3)d_0 d_1 d_4 + (c_1^2 - 2c_0 c_2)d_0 d_3 d_4 + c_0^2(-d_1 d_4^2 + d_2 d_3 d_4)}{2d_0 d_4(-d_0 d_3^2 - d_1^2 d_4 + d_1 d_2 d_3)} \quad (9)$$

Partial derivatives need to be calculated and equated to zero. The solutions can be found with the script presented at the end of this assignment. There are two solutions: for $K_R \approx 14.926$ and $T_I \approx 18.006$ the model doesn't act as a PI regulator, so the values $K_R \approx 1.417$ and $T_I \approx 8.484$ are the ones we were looking for.

1.2 Simulation of behaviour

The behavior of the closed-loop control system from the subtask [1.1] with the reference signal $r(t) = S(t)$ and the disturbance signal $z(t) = S(t - 60)$ was simulated. The obtained control signal $u(t)$ and the output signal $y(t)$ were plotted on the same figure.

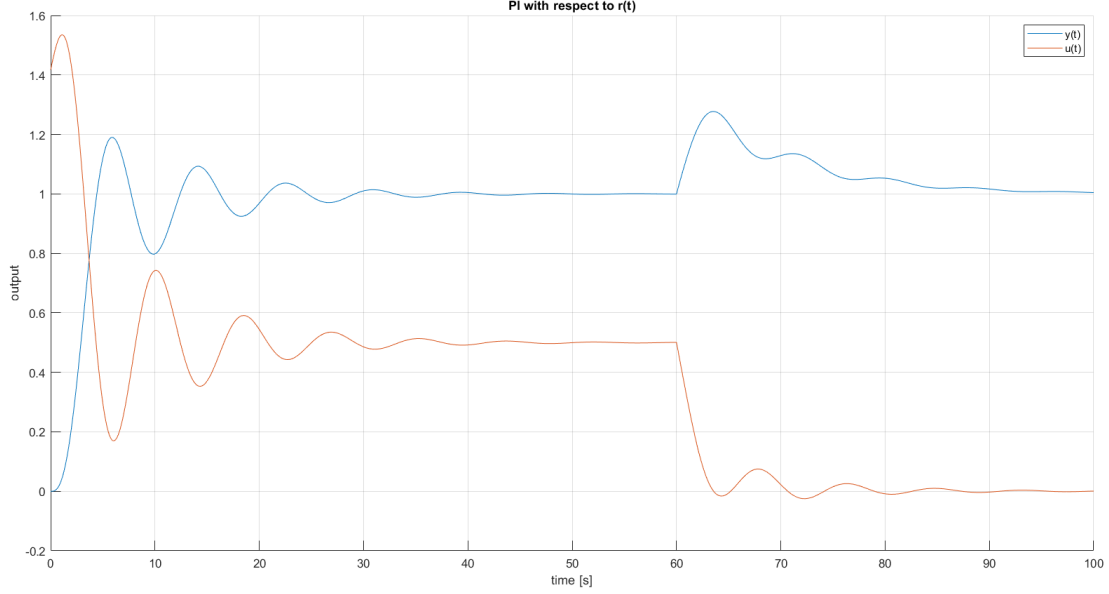


Figure 2: Comparison of $u(t)$ and $y(t)$ of the system.

1.3 Parametrizing a PI regulator with respect to $z(t)$

We assume the following substitutions:

- $G_R(s) = K_R(1 + \frac{1}{T_I s})$,
- $G_1(s) = \frac{2}{1.5s^2 + s + 1}$,
- $G_2(s) = \frac{1}{7.5s + 1}$.

The error signal with respect to a step reference signal $z(t)$ needs to be calculated:

$$E(s) = -Y(s), \quad (10)$$

$$E(s) = -\frac{G_2(s)}{1 + G_R(s)G_1(s)G_2(s)}Z(s), \quad (11)$$

$$E(s) = -\frac{\frac{1}{7.5s+1}}{1 + K_R(1 + \frac{1}{T_I s})\frac{2}{1.5s^2+s+1}\frac{1}{7.5s+1}}\frac{1}{s}, \quad (12)$$

$$E(s) = -\frac{1.5s^2 + s + 1}{(1.5s^2 + s + 1)(7.5s + 1) + 2K_R(1 + \frac{1}{T_I s})}\frac{1}{s}, \quad (13)$$

$$E(s) = -\frac{1.5T_I s^2 + T_I s + T_I}{11.25T_I s^4 + 9T_I s^3 + 8.5T_I s^2 + (1 + 2K_R)T_I s + 2K_R}, \quad (14)$$

Obtained $E(s)$ has a form $E(s) = \frac{c_0 + c_1 s + \dots + c_{n-1} s^{n-1}}{d_0 + d_1 s + \dots + d_n s^n}$ and is of the 4th order. Therefore, the ISE criterion has the form:

$$I_{3,4} = \frac{c_3^2(-d_0^2 d_3 + d_0 d_1 d_2) + (c_2^2 - 2c_1 c_3)d_0 d_1 d_4 + (c_1^2 - 2c_0 c_2)d_0 d_3 d_4 + c_0^2(-d_1 d_4^2 + d_2 d_3 d_4)}{2d_0 d_4(-d_0 d_3^2 - d_1^2 d_4 + d_1 d_2 d_3)} \quad (15)$$

Partial derivatives need to be calculated and equated to zero. The solutions can be found with the script presented at the end of this assignment. There are three solutions: for $K_R \approx 2.379$, $T_I \approx 1.137$ and $K_R \approx 10.191$, $T_I \approx 0.937$ the model doesn't act as a PI regulator (is unstable), so the values $K_R \approx 1.732$ and $T_I \approx 4.186$ are the ones we were looking for.

1.4 Simulation of behaviour II

The behavior of the closed-loop control system from the subtask [1.3] with the reference signal $r(t) = S(t)$ and the disturbance signal $z(t) = S(t - 60)$ was simulated. The obtained control signal $u(t)$ and the output signal $y(t)$ were plotted on the same figure.

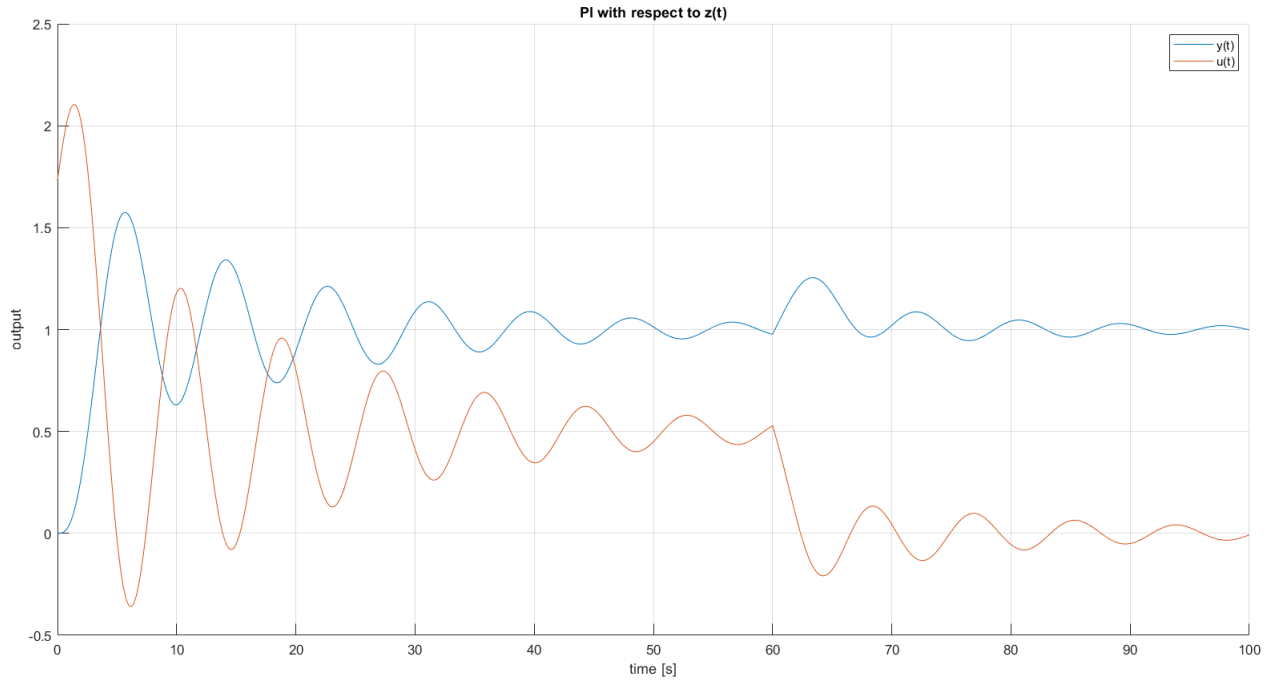


Figure 3: Comparison of $u(t)$ and $y(t)$ of the system.

1.5 Comparison of both behaviours

The results from subtasks [1.2] and [1.4] are presented below so that one figure shows the outputs signal from the process $y(t)$ and the other figure shown the control signals $u(t)$.

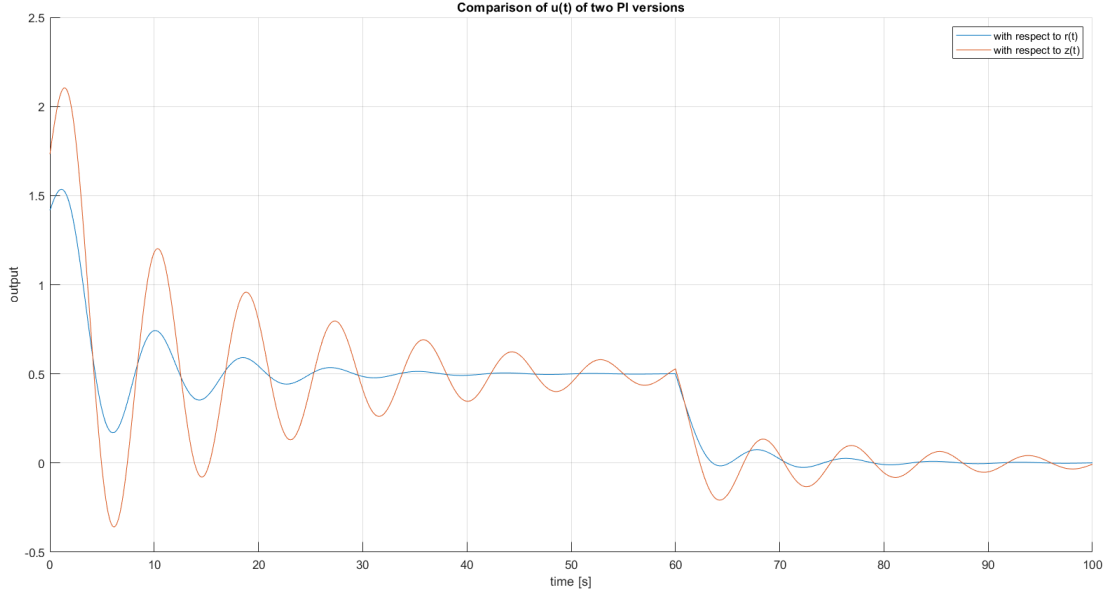


Figure 4: Comparison of inputs.

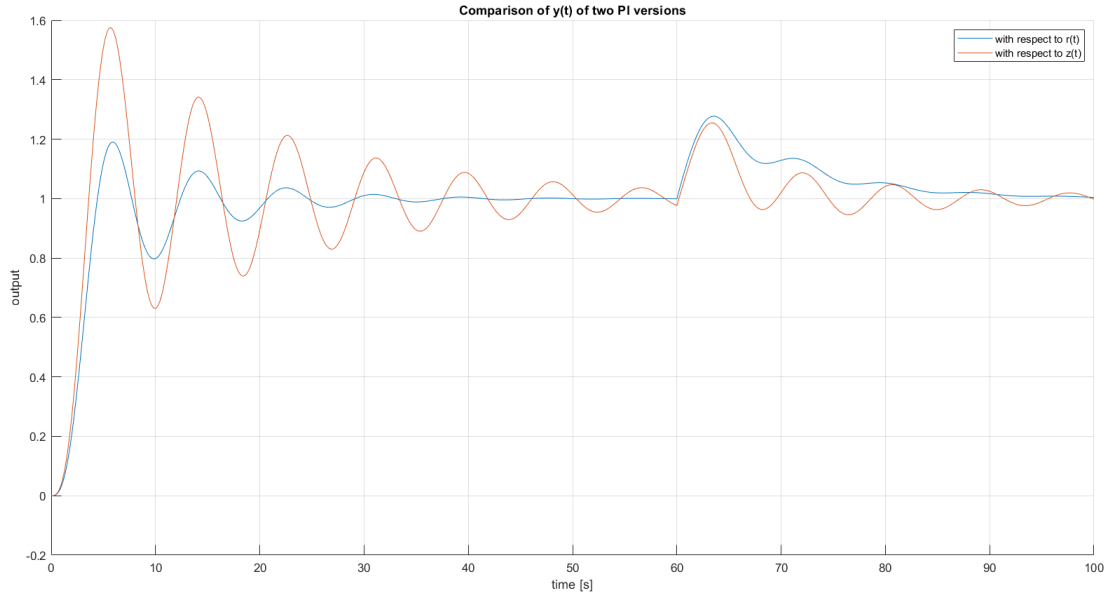


Figure 5: Comparison of outputs.

The regulator parametrized with respect to reference signal $r(t)$ has a smaller settling time with respect to the reference

signal and The regulator parametrized with respect to disturbance signal $z(t)$ has a better compensation of the disturbance.

1.6 Ziegler-Nichols method

A PI and an ideal PID regulator need to be parametrized by using the Ziegler-Nichols method based on the stability margin. Therefore, G_R is substituted with P element (K_R) and the open-loop system has to be analyzed using Nyquist criterion:

$$G(s) = \frac{2K_R}{11.25s^3 + 9s^2 + 8.5s + 1} \quad (16)$$

$$G(j\omega) = \frac{2K_R}{-11.25j\omega^3 - 9\omega^2 + 8.5j\omega + 1} \quad (17)$$

$$G(j\omega) = \frac{2K_R}{-11.25j\omega^3 - 9\omega^2 + 8.5j\omega + 1} \quad (18)$$

$$G(j\omega) = \frac{2K_R}{1 - 9\omega^2 + j\omega(8.5 - 11.25\omega^2)} \quad (19)$$

$$G(j\omega) = \frac{2K_R(1 - 9\omega^2 - j\omega(8.5 - 11.25\omega^2))}{(1 - 9\omega^2)^2 + (\omega(8.5 - 11.25\omega^2))^2} \quad (20)$$

The value of gain for which permanent oscillations are produced in the closed-loop system needs to be determined, therefore from Nyquist criterion:

$$\text{Im}\{G(j\omega)\} = 0 \iff \frac{-2K_R\omega(8.5 - 11.25\omega^2)}{(1 - 9\omega^2)^2 + (\omega(8.5 - 11.25\omega^2))^2} = 0 \iff \omega = 0 \text{ or } \omega = \sqrt{\frac{34}{45}} \quad (21)$$

For $\omega = 0$, $\text{Re}\{G(j\omega)\} = 2K_R$. It would have to be equal to -1 if we wanted to have the system on the edge of stability, but K_R has to be positive, so we cannot take this value. For $\omega = \sqrt{\frac{34}{45}}$, $\text{Re}\{G(j\omega)\} = \frac{-10K_R}{29}$. It would have to be equal to -1 if we wanted to have the system on the edge of stability, so it can be calculated that $K_R = 2.9$. At the end we obtain $K_{Rkr} = 2.9$ and $T_{kr} = \frac{2\pi}{\omega} \approx 7.2285$. Therefore, using Ziegler-Nichols rule, the following parameters are obtained:

- PI regulator: $K_R = 0.45K_{Rkr} = 1.305$, $T_I = 0.85T_{kr} \approx 6,1442$,
- ideal PID regulator: $K_R = 0.6K_{Rkr} = 1.74$, $T_I = 0.5T_{kr} \approx 3.6143$, $T_D = 0.12T_{kr} = 0.8674$.

1.7 Real PID regulator

The ideal PID regulator parameterized in task [1.6] is expanded with a small time constant $T_\nu = 0.15T_D \approx 0.1301$ so that a real PID regulator is obtained:

$$G_R(s) = 1.74 \left(1 + \frac{1}{3.6143s} + \frac{0.8674s}{1 + 0.1301s} \right). \quad (22)$$

1.8 Simulation of behaviour III

The behavior of the closed-loop control system with a PI regulator from subtask [1.6] and with a real PID regulator from task [1.7] with the reference signal $r(t) = S(t)$ and the disturbance signal $z(t) = S(t - 60)$ was simulated. The output signals $y(t)$ were plotted on one figure and the control signals $u(t)$ on the other figure.

It can be seen that PID regulator better deals with compensating the disturbance and has a smaller settling time but on the other hand has a bigger overshoot and demands more energy put into the system than PI regulator.

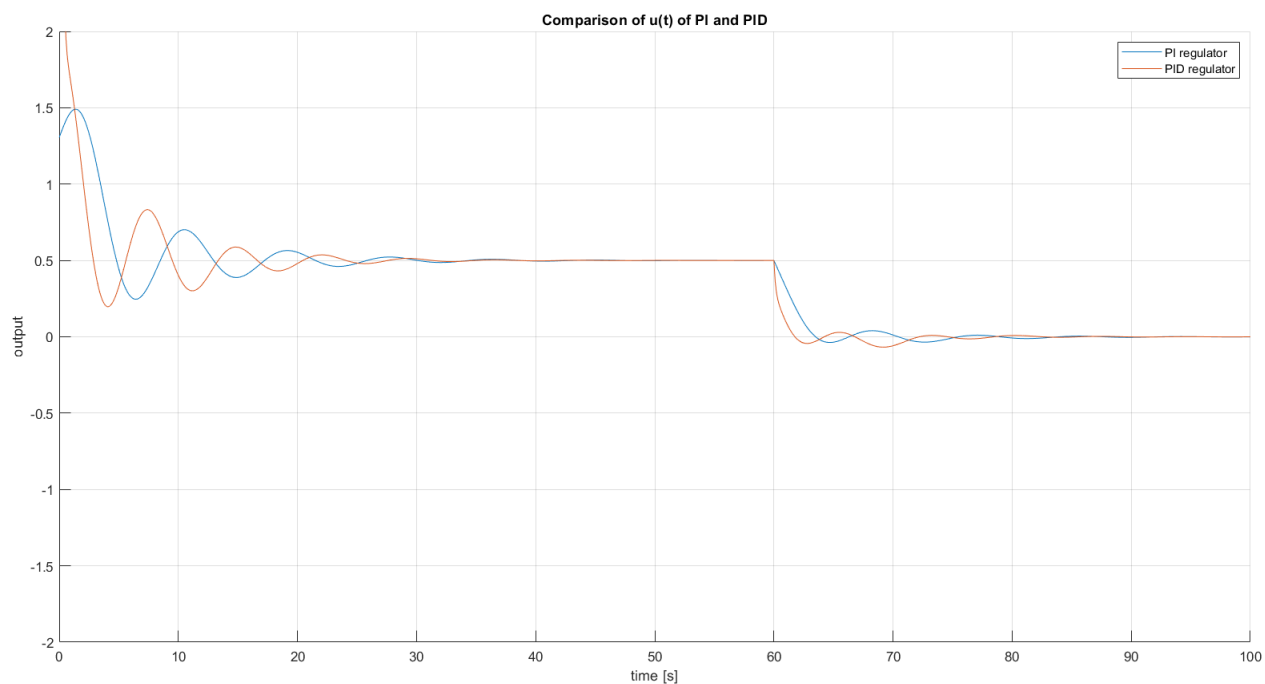


Figure 6: Comparison of $u(t)$.

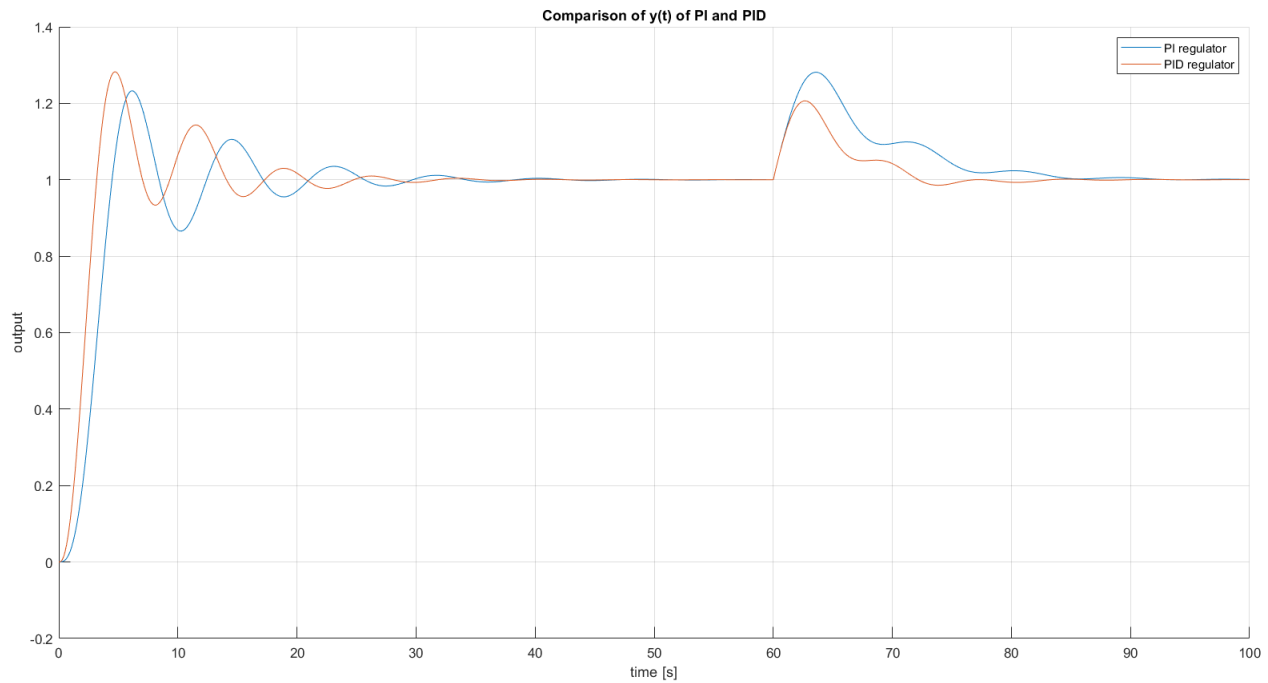


Figure 7: Comparison of $y(t)$.

1.9 Matlab script & Simulink model

```
close all;
clear all;

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%

sim_time = 100;

r_steptime = 0;
r_delta = 1;
r_initvalue = 0;

z_steptime = 60;
z_delta = 1;
z_initvalue = 0;

max_step_size = 0.1;

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
% PI element v1 - derivatives

syms T_I K_R

d_0 = 2*K_R;
```



```

d_1 = (1 + 2*K_R)*T_I;
d_2 = 8.5*T_I;
d_3 = 9*T_I;
d_4 = 11.25*T_I;

c_0 = T_I;
c_1 = 8.5*T_I;
c_2 = 9*T_I;
c_3 = 11.25*T_I;

I_4 = (c_3^2*(-d_0^2*d_3 + d_0*d_1*d_2) + (c_2^2-2*c_1*c_3)*d_0*d_1*d_4 + (c_1^2-2*c_0*c_2)*d_0*d_3*d_4 +
c_0^2*(-d_1*d_4^2 + d_2*d_3*d_4))...
/(2*d_0*d_4*(-d_0*d_3^2 - d_1^2*d_4 + d_1*d_2*d_3));

dT_I = diff(I_4, T_I);
dK_R = diff(I_4, K_R);

S = vpasolve([dT_I == 0, dK_R == 0], [T_I, K_R], [0+1e-10, inf; 0+1e-10, inf;]);

K_R1 = double(S.K_R(1));
K_R2 = double(S.K_R(2));

T_I1 = double(S.T_I(1));
T_I2 = double(S.T_I(2));

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
% PI element v1 - figures
K_r = K_R1;
T_I = T_I1;
T_D = 0;
T_V = 0;

sim('model1');
figure(1); hold on;
plot(ans.t, ans.y);
plot(ans.t, ans.u);
grid on;
xlabel('time [s]');
ylabel('output');
legend('y(t)', 'u(t)');
title('PI with respect to r(t)');

figure(3); hold on;
plot(ans.t, ans.y);

figure(4); hold on;
plot(ans.t, ans.u);

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
%PI element v2 - derivatives
syms T_I K_R

d_0 = 2*K_R;
d_1 = (1 + 2*K_R)*T_I;
d_2 = 8.5*T_I;

```

```

d_3 = 9*T_I;
d_4 = 11.25*T_I;

c_0 = -T_I;
c_1 = -T_I;
c_2 = -1.5*T_I;
c_3 = sym('0');

I_4 = (c_3^2*(-d_0^2*d_3 + d_0*d_1*d_2) + (c_2^2-2*c_1*c_3)*d_0*d_1*d_4 + (c_1^2-2*c_0*c_2)*d_0*d_3*d_4 +
c_0^2*(-d_1*d_4^2 + d_2*d_3*d_4))...
/(2*d_0*d_4*(-d_0*d_3^2 - d_1^2*d_4 + d_1*d_2*d_3));

dT_I = diff(I_4, T_I);
dK_R = diff(I_4, K_R);

S = vpasolve([dT_I == 0, dK_R == 0], [T_I, K_R], [0+1e-10, inf; 0+1e-10, inf;]);

K_R1 = double(S.K_R(1));
K_R2 = double(S.K_R(2));
K_R3 = double(S.K_R(3));

T_I1 = double(S.T_I(1));
T_I2 = double(S.T_I(2));
T_I3 = double(S.T_I(3));

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
% PI element v2 - figures
K_r = K_R2;
T_I = T_I2;
T_D = 0;
T_V = 0;

sim('model1');
figure(2); hold on;
plot(ans.t, ans.y);
plot(ans.t, ans.u);
grid on;
xlabel('time [s]');
ylabel('output');
legend('y(t)', 'u(t)');
title('PI with respect to z(t)');

figure(3); hold on;
plot(ans.t, ans.y);

figure(4); hold on;
plot(ans.t, ans.u);

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
% comparison of v1 and v2

figure(3); hold on; grid on;
xlabel('time [s]');
ylabel('output');
legend('with respect to r(t)', 'with respect to z(t)');
title('Comparison of y(t) of two PI versions');

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figure(4); hold on; grid on;
xlabel('time [s]');
ylabel('output');
legend('with respect to r(t)', 'with respect to z(t)');
title('Comparison of u(t) of two PI versions');

%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
% PI element v3
K_r = 1.305;
T_I = 6.1442;
T_D = 0;
T_V = 0;

sim('model1');
figure(5); hold on;
plot(ans.t, ans.y);

figure(6); hold on;
plot(ans.t, ans.u);

% PID element
K_r = 1.74;
T_I = 3.6143;
T_D = 0.8674;
T_V = 0.1301;

sim('model1');
figure(5); hold on;
plot(ans.t, ans.y);

figure(6); hold on;
plot(ans.t, ans.u);

figure(5); hold on; grid on;
xlabel('time [s]');
ylabel('output');
legend('PI regulator', 'PID regulator');
title('Comparison of y(t) of PI and PID');

figure(6); hold on; grid on;
ylim([-2 2]);
xlabel('time [s]');
ylabel('output');
legend('PI regulator', 'PID regulator');
title('Comparison of u(t) of PI and PID');

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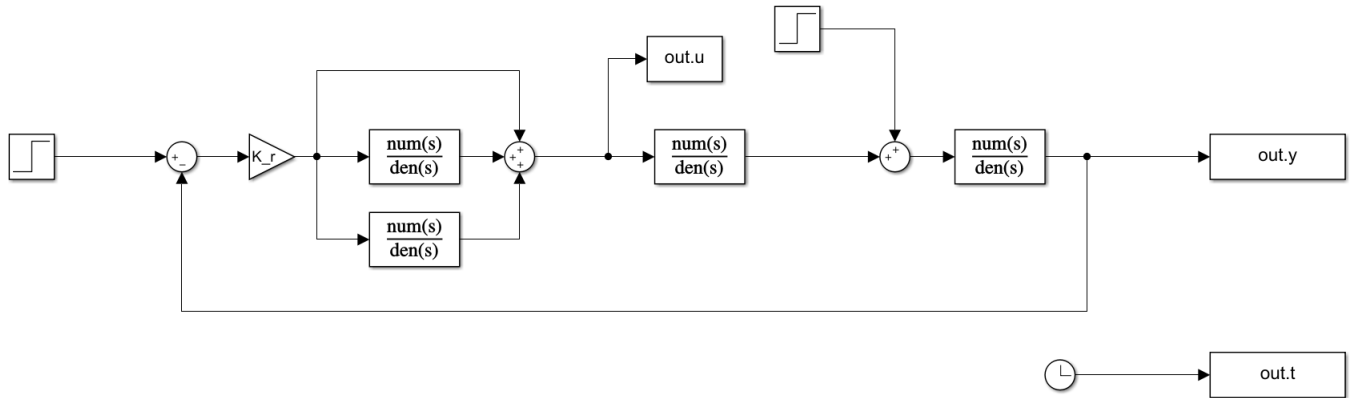


Figure 8: Simulink model.